

# Estimating real-world energy savings of an electric city bus with in-wheel motors versus central drive configuration



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- RET: *Rotterdamse Elektrische Tram*
  - Public transport organisation
  - Founded in 1927
  - In 2022: 97 zero emission city buses



Claim from an E-driveline supplier:  
*“Application of in-wheel motors reduces the energy consumption by 15%”*



## Problem statement:

- Can the claim be confirmed without running tests?
- We also like to better understand the origin of the energy savings.

## Problem approach:

- Execution of a drive line energy consumption comparison by means of 1D software Simcenter AMESim



Source: <https://www.tech-story.net>

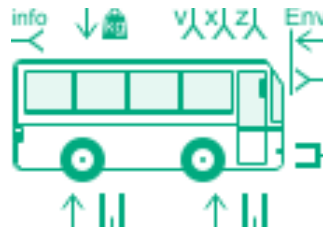
Claim from an E-driveline supplier:

*“Application of in-wheel motors reduces the energy consumption by 15%”*


# The modeling approach



Focus on the **largest road load related energy losses**,  
and with a **sufficient level of detail**  
by making use of commercially available **1D software**.

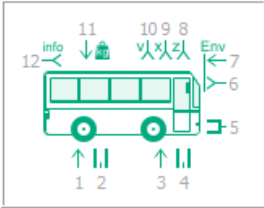


# Model setup


Change Parameters

?
X

Submodel



drv\_vehicle\_1 [DRVEH01]

external variable:

variable load vehicle with or without slip

Image >>

Parameters

Title	Value	Unit	Tags
# vehicle linear velocity at port 5	0	m/s	
# vehicle position at port 5	0	km	
# vehicle linear displacement at port 12	0	m	
vehicle index	2		
vehicle configuration	road		
longitudinal slip configuration	without slip		
icon	bus		
total vehicle mass	mVehicle	kg	
<div> <div> aerodynamic and rolling parameters </div> <div> wheel characteristics </div> <div> brake characteristics </div> </div>			

Save

Load

Default value

Max. value

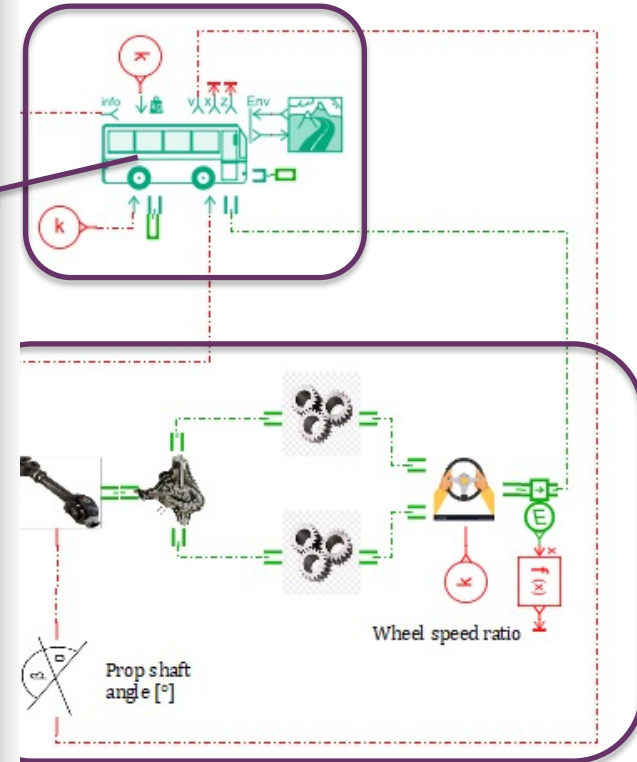
Reset title

Min. value

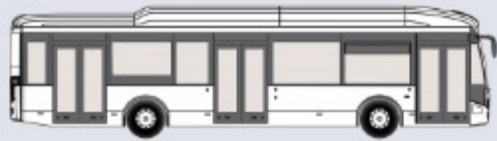
Help

Close

Options >>



- Publicly accessible information
- Literature



Citea SLF-120



40\_020266\_01

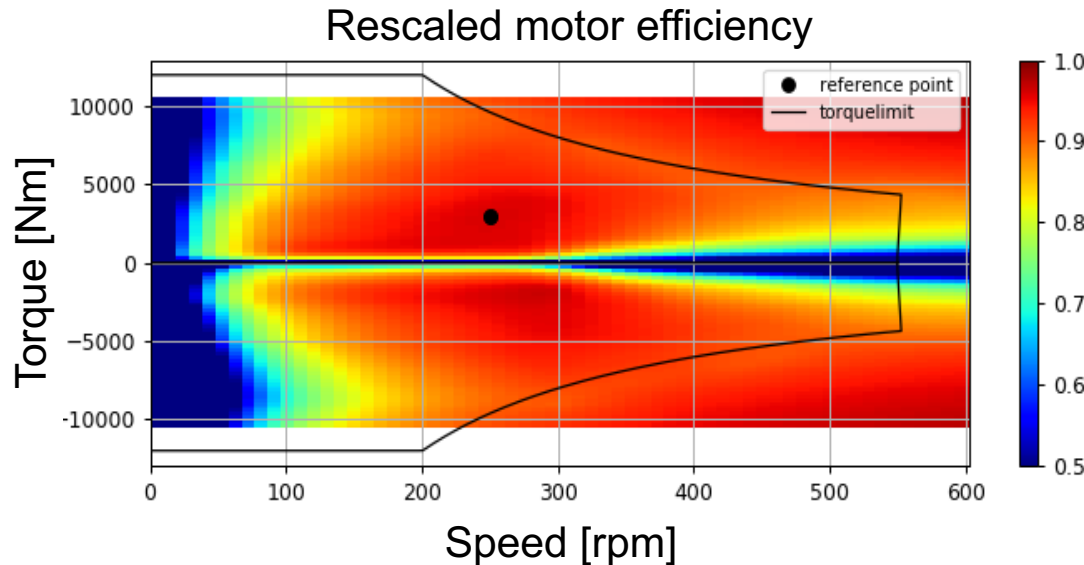
## Typical road load characteristics

	12 m urban bus
Frontal area [m <sup>2</sup> ]	8,4
Vehicle mass, empty [kg]	13250
Vehicle mass in calculations [kg]	14500
Air drag resistance coefficient Cd [-]	0,7
Number of gears	Fixed ratio
Rolling losses resistance coefficient Crr [-]	0,006
Tyre type	275/70R22,5

Source: <https://www.vdlbuscoach.com/en/products/citea/citea-slf-slfa-electric>  
 The coefficient values are estimated and have not been supplied by VDL

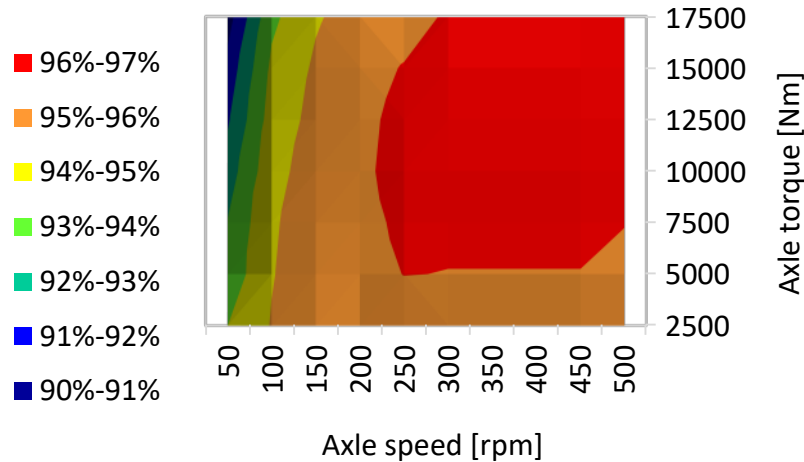
Source: Maintenance instruction AV133  
<http://shop.pwt.ca/DRT/Vendors%20Manuals/04000%20Axles%20-%20ZF/4472.758.101b%20Operating%20Instructions%20AV-110%20&%20AV-133.pdf>

- The current motor + inverter efficiency map is unknown, so a default map from AMESim is taken, with the reference point estimated for this application

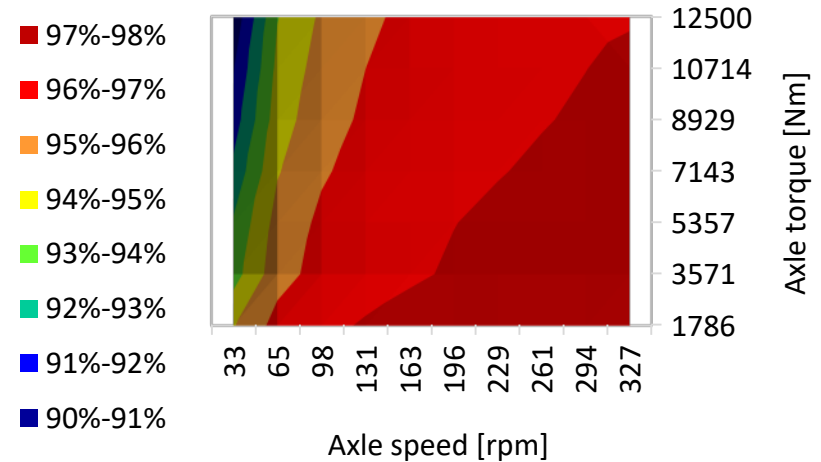


- The in-wheel motor efficiency map lies 3% lower than the central drive one

Tested MTV Axle Efficiency Map 75W-90



Simulated Axle Efficiency Map VDL bus



Comparison between measured efficiency from literature [10] (left) and simulated Central Drive line (CDR) efficiency (right)



Trendwise comparison confirms the modeling approach



# Simulation results

Table 3: Cycle energy consumption for different driveline configurations

Cycle energy consumption [kWh/km]	Central drive (CDR)	In-wheel (IW)	IW compared to CDR
Dutch Urban Bus cycle [11]	1.04	0.93	-11%
Braunschweig cycle [11]	1.00	0.90	-10%
SORT heavy urban cycle [12]	1.21	1.10	-9%
SORT suburban cycle [12]	1.21	1.12	-8%
<i>Average</i>	<i>1.12</i>	<i>1.02</i>	<b>-9%</b>

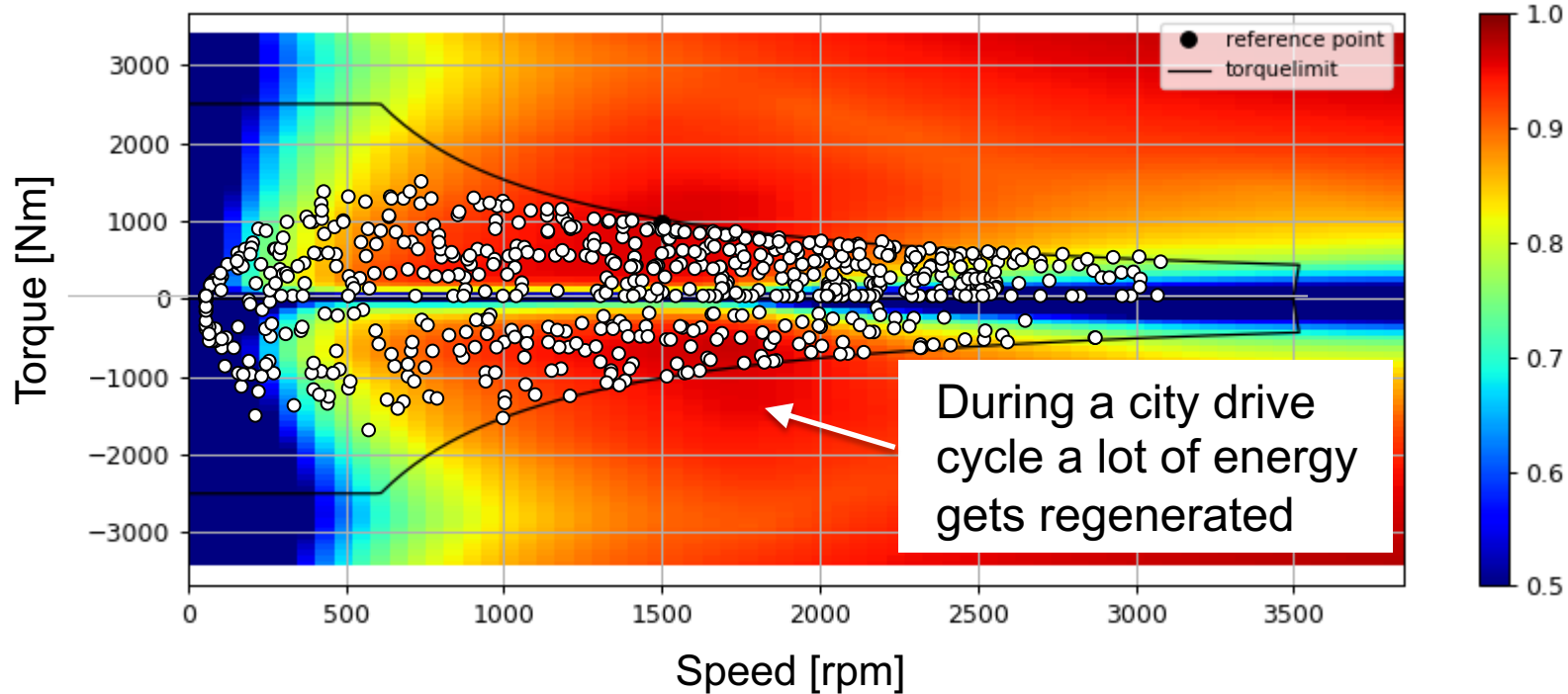
These results are based on the assumption that the In-Wheel motor and inverter together have even a 3% lower efficiency than the Central Drive ones.



**It's a significant improvement. Why?**

# Simulation results

Rescaled motor efficiency



Drive line losses very negatively impact the drive range

Simulation results show 8 ~11% better efficiency for IW compared to CDR

- The 15% claim is somewhat optimistic
- This is based on non validated model simulation (i.e. indicative)

The choice of drive line configuration also depends on other criteria.



# Thank you for your attention

## Acknowledgement

The generic primary model building activity in this study were part of the RAAK PRO VETIS project, funded by the Dutch Taskforce for Applied Research SIA. The detailed drivetrain modelling activities and reporting were funded by the Automotive Center of Expertise (ACE). The authors wish to thank both funding organizations for their support.

